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A MATHEMATICAL ANALYSIS OF THE STABILITY  
OF A LUNAR FLYING VEHICLE WITH MANUAL CONTROLS

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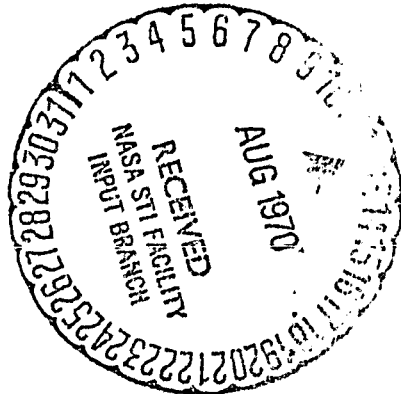
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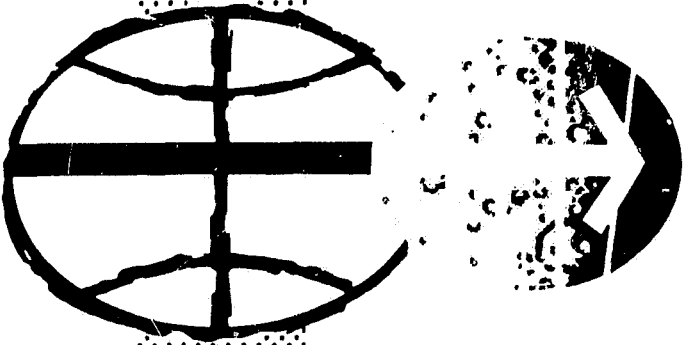
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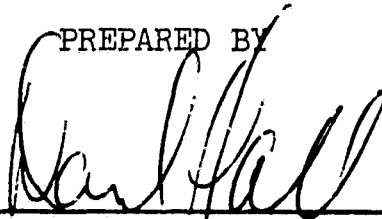
MANNED SPACECRAFT CENTER  
HOUSTON, TEXAS

September 5, 1968

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A MATHEMATICAL ANALYSIS OF THE STABILITY  
OF A LUNAR FLYING VEHICLE WITH MANUAL CONTROLS

PREPARED BY




---

David Hall

Flight Performance Section

AUTHORIZED FOR DISTRIBUTION



for Maxime A. Faget

Director of Engineering and Development

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

MANNED SPACECRAFT CENTER

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# A MATHEMATICAL ANALYSIS OF THE STABILITY OF A LUNAR FLYING VEHICLE WITH MANUAL CONTROLS

By David Hall

## SUMMARY

Equations of motion for a four-degree-of-freedom mathematical model of the lunar flying vehicle are developed and used to study the stability of the autonomous vehicle. The human pilot describing function is used to represent the control system and derive the stability criteria for the closed-loop attitude control system.

## INTRODUCTION

The system under investigation is a free-flight manually controlled vehicle operating in the lunar environment. A mathematical model of the vehicle dynamics and the pilot control function is used to study the system stability. The system is represented by the block diagram in figure 1.

The vehicle consists of a platform which supports the pilot, propellant tanks, engine, landing gear, and equipment. Representative configurations of thrust vector-controlled and kinesthetic-controlled vehicles will be discussed. The coordinates which describe the motion of the bodies are shown in figure 2.

The vehicle dynamics are based on a four-degree-of-freedom mathematical model of plane motion, which considers two rigid bodies connected at a point which will be referred to as the gimbal point. Two coordinates  $X$  and  $Z$  locate the center of mass of one body in the plane. One coordinate  $\theta$  determines the angular position of the body with respect to vertical, and the fourth coordinate  $\beta$  determines the angular position of the second body with respect to vertical.

For the purposes of this study, several assumptions were made which simplify the analysis but retain the fidelity of the model.

1. Flight time is small (approximately 4 minutes).
2. Mass, center of mass, and inertia of each body are constant.
3. Pilot describing function is time invariant and can be represented by the Laplace transform.
4. Pilot control force is applied normal to the thrust vector which is along the coordinate axis of body 2.
5. Pilot control force reaction is applied at the center of mass of body 1, parallel to the  $x_{b_1}$  body axis.

The equations of motion are developed using Lagrange's equation and solved for the angular accelerations. The Jacobian of the solution allows the equations to be written in state variable matrix notation. Routh's criteria are used to determine the stability requirements for the vehicle characteristic equation.

Vehicle control is represented by the experimentally determined human pilot describing function. The closed-loop system characteristic equation is derived and Hurwitz criteria used to study the effect on stability of varying the vehicle and control function parameters.

The mathematical model has been programed for the digital computer and particular solutions have been generated to verify the analytical results. Although this model is not an analog of the manually controlled vehicle with variable mass characteristics, it will hopefully provide some insight into the stability and control of vehicles in this class. The philosophy is that an approximate model will do in the beginning and that we will enhance our theoretical understanding by attempting engineering applications and learning from them.

Work related to this analysis is presented in the Bibliography.

#### SYMBOLS

$a_{ij}$  matrix element

$b_i$  coefficient of  $s^n$  in characteristic equation

$c$	damping coefficient
$D$	dissipation function
$e$	system error
$e$	base of natural logarithms
$F_c$	control force
$G$	transfer function
$g$	acceleration of gravity
$H$	Hurwitz matrix
$H_n$	principal minor of Hurwitz matrix
$I$	moment of inertia
$J$	Jacobian matrix
$K_p$	pilot gain
$k$	spring coefficient
$L_1$	directed length from body 1 center of mass to gimbal
$L_2$	directed length from gimbal to body 2 center of mass
$L_c$	directed length from gimbal to application of control force
$M$	mass
$Q$	generalized force
$q$	generalized coordinate
$s$	Laplace transform operator
$T$	engine thrust
$T$	kinetic energy function



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$T_I$	general lag time constant
$T_L$	general leadtime constant
$T_N$	lag time constant approximation of the neuromuscular system
$t$	time
$V$	potential energy function
$X$	coordinate of body 1 center of mass
$x_b$	body axis
$Y_c$	controlled element transfer function
$Y_p$	pilot describing function
$Z$	coordinate of body 1 center of mass
$z_b$	body axis
$\beta$	coordinate of body 2 angular position
$\Delta$	determinant of matrix
$\delta$	gimbal angle
$\zeta_N$	damping ratio of the second-order component of the neuromuscular system
$\theta$	coordinate of body 1 angular position
$\tau$	pilot reaction time lag
$\omega_N$	undamped natural frequency of the second-order component of the neuromuscular system

Subscripts:

1	body 1
2	body 2

b	body
c	control
i	input
N	neuromuscular system
n	integer
p	pilot

Operators:

$\frac{d( )}{dq}$  derivative of ( ) with respect to q

$\frac{\partial( )}{\partial q}$  partial derivative of ( ) with respect to q

$(\dot{\phantom{x}})$  derivative of ( ) with respect to time

$(\ddot{\phantom{x}})$  second derivative of ( ) with respect to time

$\approx$  approximately equal to

## ANALYSIS

### Vehicle Equations of Motion

The analysis of vehicle dynamics is limited to plane motion to allow the effect of vehicle design on stability to be emphasized without unnecessary complexity. This results in a system with four degrees of freedom and consisting of two rigid bodies connected at a point.

The vehicle equations of motion can be derived using Lagrange's equation

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_n} \right) - \frac{\partial T}{\partial q_n} + \frac{\partial D}{\partial \dot{q}_n} + \frac{\partial V}{\partial q_n} = Q_n \quad (1)$$

The functions are

$$\begin{aligned}
 T = & \frac{1}{2} M_1 (\dot{X}^2 + \dot{Z}^2) + \frac{1}{2} I_1 \dot{\theta}^2 + \frac{1}{2} M_2 [\dot{X}^2 + \dot{Z}^2 + L_1^2 \dot{\theta}^2 \\
 & + L_2^2 \dot{\beta}^2 + 2L_1 \dot{\theta} (\dot{X} \cos \theta - \dot{Z} \sin \theta) \\
 & + 2L_2 \dot{\beta} (\dot{X} \cos \beta - \dot{Z} \sin \beta) + 2L_1 L_2 \dot{\theta} \dot{\beta} \cos(\theta - \beta)] \\
 & + \frac{1}{2} I_2 \dot{\beta}^2
 \end{aligned}$$

$$D = \frac{1}{2} c (\dot{\theta} - \dot{\beta})^2$$

$$V = M_1 g Z + M_2 g (Z + L_1 \cos \theta + L_2 \cos \beta) + \frac{1}{2} k (\theta - \beta)^2$$

Let  $q_1 = \theta$

$$q_2 = \beta$$

$$q_3 = X$$

$$q_4 = Z$$

Then  $Q_1 = -TL_1 \sin(\theta - \beta)$

$$Q_2 = -F_c L_c$$

$$Q_3 = T \sin \beta + F_c \cos \theta$$

$$Q_4 = T \cos \beta - F_c \sin \theta$$

Differentiating, substituting into the Lagrange equation, and rearranging the equations into matrix form give the equations of motion

$$\begin{bmatrix} I_1 + M_{21} L_1^2 & M_{21} L_1 L_2 \cos(\theta - \beta) & M_{21} L_1 \cos \theta & -M_{21} L_1 \sin \theta \\ M_{21} L_1 L_2 \cos(\theta - \beta) & I_2 + M_{22} L_2^2 & M_{22} L_2 \cos \beta & -M_{22} L_2 \sin \beta \\ M_{21} L_1 \cos \theta & M_{22} L_2 \cos \beta & M_1 + M_2 & 0 \\ -M_{21} L_1 \sin \theta & -M_{22} L_2 \sin \beta & 0 & M_1 + M_2 \end{bmatrix} \begin{bmatrix} \ddot{\theta} \\ \ddot{\beta} \\ \ddot{X} \\ \ddot{Z} \end{bmatrix} = \begin{bmatrix} -k(\theta - \beta) - c(\dot{\theta} - \dot{\beta}) - M_{21} L_1 L_2 \ddot{\beta} \sin(\theta - \beta) + M_2 g L_1 \sin \theta - T L_1 \sin(\theta - \beta) \\ k(\theta - \beta) + c(\dot{\theta} - \dot{\beta}) + M_{21} L_1 L_2 \ddot{\theta} \sin(\theta - \beta) + M_2 g L_2 \sin \beta - F_c L_c \\ M_{21} L_1 \ddot{\theta} \sin \theta + M_{22} L_2 \ddot{\beta} \sin \beta + T \sin \beta + F_c \cos \theta \\ M_{21} L_1 \ddot{\theta} \cos \theta + M_{22} L_2 \ddot{\beta} \cos \beta - (M_1 + M_2) g + T \cos \beta - F_c \sin \theta \end{bmatrix} \quad (2)$$

The solution of equation (2) will be of the form

$$\ddot{\theta} = f_1(\theta, \dot{\theta}, \beta, \dot{\beta}, F_c)$$

$$\ddot{\beta} = f_2(\theta, \dot{\theta}, \beta, \dot{\beta}, F_c)$$

$$\ddot{x} = f_3(\theta, \dot{\theta}, \beta, \dot{\beta}, F_c)$$

$$\ddot{z} = f_4(\theta, \dot{\theta}, \beta, \dot{\beta}, F_c)$$

Since the accelerations are independent of the linear position and velocity, the stability of the rotational motion may be investigated separately. Consequently, equation (2) was solved for the angular accelerations by premultiplying by the inverse of the coefficient matrix. Letting  $\Delta$  represent the determinant of the coefficient matrix, the angular accelerations are

$$\begin{aligned} \Delta \ddot{\theta} = & -k(M_1 + M_2) \left\{ (M_1 + M_2) I_2 + M_1 M_2 L_2 [L_2 + L_1 \cos(\theta - \beta)] \right\} (\theta - \beta) \\ & - c(M_1 + M_2) \left\{ (M_1 + M_2) I_2 + M_1 M_2 L_2 [L_2 + L_1 \cos(\theta - \beta)] \right\} (\dot{\theta} - \dot{\beta}) \\ & - M_1^2 M_2^2 L_1^2 L_2^2 \cos(\theta - \beta) \sin(\theta - \beta) \dot{\theta}^2 \\ & - M_1 M_2 L_1 L_2 (M_1 I_2 + M_1 M_2 L_2^2 + M_2 I_2) \sin(\theta - \beta) \dot{\beta}^2 \\ & - M_1 T L_1 (M_1 I_2 + M_1 M_2 L_2^2 + M_2 I_2) \sin(\theta - \beta) \\ & - M_2 F_c L_1 \left\{ (M_1 + M_2) [I_2 - M_1 L_2 L_c \cos(\theta - \beta)] \right. \\ & \left. + M_1 M_2 L_2^2 \sin^2(\theta - \beta) \right\} \end{aligned} \quad (3)$$

$$\begin{aligned}
\Delta \ddot{\beta} = & k(M_1 + M_2) \left\{ (M_1 + M_2) I_1 + M_1 M_2 L_1 [L_1 + L_2 \cos(\theta - \beta)] \right\} (\theta - \beta) \\
& + c(M_1 + M_2) \left\{ (M_1 + M_2) I_1 + M_1 M_2 L_1 [L_1 + L_2 \cos(\theta - \beta)] \right\} (\dot{\theta} - \dot{\beta}) \\
& + M_1 M_2 L_1 L_2 (M_1 I_1 + M_1 M_2 L_1^2 + M_2 I_1) \sin(\theta - \beta) \dot{\theta}^2 \\
& + M_1^2 M_2^2 L_1^2 L_2^2 \cos(\theta - \beta) \sin(\theta - \beta) \dot{\beta}^2 \\
& + M_1^2 M_2^2 L_1^2 L_2^2 \cos(\theta - \beta) \sin(\theta - \beta) \\
& - (M_1 + M_2) F_c \left\{ M_2 I_1 [L_c + L_2 \cos(\theta - \beta)] + M_1 L_c (I_1 + M_2 L_1^2) \right\} \quad (4)
\end{aligned}$$

where

$$\begin{aligned}
\Delta = & (M_1 + M_2) \left[ I_2 M_2 (I_1 + M_1 L_1^2) + I_1 M_1 (I_2 + M_2 L_2^2) \right] \\
& + M_1^2 M_2^2 L_1^2 L_2^2 \sin^2(\theta - \beta) \quad (5)
\end{aligned}$$

#### Vehicle Stability

Equilibrium is defined as that position of the variables for which the derivatives of the variables are zero. From equations (3), (4), and (5), the equilibrium position for the rotational motion of the autonomous vehicle is  $(\theta - \beta) = 0$ . To study the vehicle stability in the neighborhood of equilibrium, the Jacobian is used to obtain a linear approximation of the acceleration equations.

The Jacobian is

$$J = \begin{bmatrix} \frac{\partial f_1}{\partial q_1} & \frac{\partial f_1}{\partial q_2} & \frac{\partial f_1}{\partial q_3} & \frac{\partial f_1}{\partial q_4} \\ \frac{\partial f_2}{\partial q_1} & \frac{\partial f_2}{\partial q_2} & \frac{\partial f_2}{\partial q_3} & \frac{\partial f_2}{\partial q_4} \\ \frac{\partial f_3}{\partial q_1} & \frac{\partial f_3}{\partial q_2} & \frac{\partial f_3}{\partial q_3} & \frac{\partial f_3}{\partial q_4} \\ \frac{\partial f_4}{\partial q_1} & \frac{\partial f_4}{\partial q_2} & \frac{\partial f_4}{\partial q_3} & \frac{\partial f_4}{\partial q_4} \end{bmatrix}$$

where  $f_1 = \dot{\theta}$

$f_2 = \ddot{\theta}$

$f_3 = \dot{\beta}$

$f_4 = \ddot{\beta}$

The equations for rotational motion may then be written in state variable matrix form

$$\begin{bmatrix} \dot{\theta} \\ \ddot{\theta} \\ \dot{\beta} \\ \ddot{\beta} \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ a_{21} & a_{22} & a_{23} & a_{24} \\ 0 & 0 & 0 & 1 \\ a_{41} & a_{42} & a_{43} & a_{44} \end{bmatrix} \begin{bmatrix} \theta \\ \dot{\theta} \\ \beta \\ \dot{\beta} \end{bmatrix} + \begin{bmatrix} 0 \\ a_{25} \\ 0 \\ a_{45} \end{bmatrix} F_c \quad (6)$$

where

$$\left. \begin{aligned}
 a_{21} = -a_{23} &= -\frac{1}{\Delta} \left\{ k(M_1 + M_2) \left[ (M_1 + M_2)I_2 + M_1M_2L_2(L_2 + L_1) \right] \right. \\
 &\quad \left. + M_1TL_1(M_1I_2 + M_1M_2L_2^2 + M_2I_2) \right\} \\
 a_{22} = -a_{24} &= -\frac{1}{\Delta} c(M_1 + M_2) \left[ (M_1 + M_2)I_2 + M_1M_2L_2(L_2 + L_1) \right] \\
 a_{41} = -a_{43} &= \frac{1}{\Delta} \left\{ k(M_1 + M_2) \left[ (M_1 + M_2)I_1 + M_1M_2L_1(L_1 + L_2) \right] \right. \\
 &\quad \left. + M_1^2M_2TL_1^2L_2 \right\} \\
 a_{42} = -a_{44} &= \frac{1}{\Delta} c(M_1 + M_2) \left[ (M_1 + M_2)I_1 + M_1M_2L_1(L_1 + L_2) \right] \\
 \Delta &= (M_1 + M_2) \left[ I_2M_2(I_1 + M_1L_1^2) + I_1M_1(I_2 + M_2L_2^2) \right]
 \end{aligned} \right\} (7)$$

The adequacy of the linear approximation of the equations of motion is illustrated by figures 3 and 4 which were obtained by numerical integration of equations (2) and (6), respectively.

The third column of  $[A]$  is the negative of the first column; consequently, the matrix is singular and the system has infinitely many equilibrium states. This agrees with the statement that the vehicle is in equilibrium for  $\theta = \beta$ .

The vehicle characteristic equation is obtained by

$$\det[s[I] - [A]] = 0$$

where  $[I]$  is the identity matrix and  $[A]$  is the coefficient matrix.



The characteristic equation is

$$\begin{aligned}
 s^4 + (-a_{22} - a_{44})s^3 + (a_{22}a_{44} - a_{21} - a_{24}a_{42} - a_{43})s^2 \\
 + (a_{21}a_{44} - a_{24}a_{41} + a_{43}a_{22} - a_{23}a_{42})s \\
 + (a_{43}a_{21} - a_{23}a_{41}) = 0
 \end{aligned}$$

As a result of the relationships between the  $a_{ij}$  given above, the last two coefficients are zero, and part of the coefficient of  $s^2$  cancels.

The characteristic equation reduces to

$$s^4 + (-a_{22} - a_{44})s^3 + (-a_{21} - a_{43})s^2 = 0$$

The stability of the vehicle may be investigated by Routh's criteria. The Routhian array for a fourth-order system is

$s^4$	$b_0$	$b_2$	$b_4$
$s^3$	$b_1$	$b_3$	
$s^2$	$b_2 - \frac{b_0 b_3}{b_1}$	$b_4$	
$s^1$	$b_3 - \frac{b_1 b_4}{b_2 - \frac{b_0 b_3}{b_1}}$		
$s^0$	$b_4$		

where the  $b_i$  refer to the coefficient of  $s^{4-i}$  in the characteristic equation. Routh's criteria state that the number of roots of the characteristic equation with positive real parts is equal to the number of changes of sign of the terms in the first column of the Routhian array.

For the vehicle characteristic equation

$$b_0 = 1$$

$$b_3 = b_4 = 0$$

and the Routhian array is

$$\begin{array}{c|cc} s^4 & 1 & b_2 \\ s^3 & b_1 & \\ s^2 & b_2 & \\ s^1 & 0 & \\ s^0 & 0 & \end{array}$$

The zero in the  $s^1$  row indicates that there are roots that are the negatives of each other. From the auxiliary equation

$$b_2 s^2 = 0$$

both roots are zero. There will be no changes of sign in the first column and, therefore, no roots with positive real parts if

$$b_1 > 0$$

and

$$b_2 > 0$$

In terms of the vehicle parameters

$$b_1 = -a_{22} - a_{44} = \frac{c \left[ (M_1 + M_2)(I_1 + I_2) + M_1 M_2 (L_1 + L_2)^2 \right]}{I_2 M_2 (I_1 + M_1 L_1^2) + I_1 M_1 (I_2 + M_2 L_2^2)} \quad (8)$$

$$b_2 = -a_{21} - a_{43} = \frac{k(M_1 + M_2) \left[ (M_1 + M_2)(I_1 + I_2) + M_1 M_2 (L_1 + L_2)^2 \right] + M_1 T L_1 \left[ (M_1 + M_2) I_2 + M_1 M_2 L_2 (L_1 + L_2) \right]}{(M_1 + M_2) \left[ I_2 M_2 (I_1 + M_1 L_1^2) + I_1 M_1 (I_2 + M_2 L_2^2) \right]} \quad (9)$$

Since all the terms in equation (8) are positive,  $b_1$  is always greater than zero. The denominator of equation (9) is positive; therefore, the sign of  $b_2$  will depend on the sign and magnitude of the terms in the numerator.

For example, with the gimbal point below the center of mass,  $L_1$  is negative (fig. 2 and definition of symbols). Assuming  $L_2$  is positive, the following equation must be satisfied for nondivergent oscillations.

$$\begin{aligned} k(M_1 + M_2) \left[ (M_1 + M_2)(I_1 + I_2) + M_1 M_2 (L_2 - L_1)^2 \right] \\ \geq M_1 T L_1 \left[ (M_1 + M_2) I_2 + M_1 M_2 L_2 (L_2 - L_1) \right] \end{aligned} \quad (10)$$

In the kinesthetic-controlled vehicle concept, it is assumed that the pilot acts as a rigid body pivoted at his ankles. Therefore, the mass, inertia, and characteristic length of body 1 are fixed, and the design of the platform is constrained to be compatible with the stability criteria.

The existence of a double root at  $s = 0$  indicates that the origin is unstable. This can be understood by examining the block diagram of the vehicle equations of motion (fig. 5). The initial conditions

$$\dot{\theta}(0) = \dot{\beta}(0) \neq 0$$

will produce a linear and equal increase in  $\theta$  and  $\beta$ .

For zero input, the solution of equation (6) is

$$[q(s)] = [s[I] - [A]]^{-1} [q(0)] \quad (11)$$

where

$$[s[I] - [A]]^{-1}$$

is the Laplace transform of the fundamental matrix.

For the vehicle system, the matrix is

$$[s[I] - [A]]^{-1} = \frac{\begin{bmatrix} s^3 + (-a_{22} - a_{44})s^2 - a_{43}s^2 - a_{44}s - a_{43} & a_{23}s + (a_{24}a_{43} - a_{23}a_{44}) & a_{24}s + a_{23} \\ + (a_{22}a_{43} - a_{23}a_{42}) & & \\ a_{21}s^2 + (a_{24}a_{41} - a_{21}a_{44})s & s^3 - a_{44}s^2 - a_{43}s & a_{24}s^2 + a_{23}s \\ a_{41}s + (a_{21}a_{42} - a_{41}a_{22}) & a_{42}s + a_{41} & s^3 + (-a_{22} - a_{44})s^2 - a_{21}s^2 - a_{22}s - a_{21} \\ & & + (a_{21}a_{44} - a_{24}a_{41}) \end{bmatrix}}{s^2 \left[ s^2 + (-a_{22} - a_{44})s + (-a_{21} - a_{43}) \right]}$$

(12)

The matrix shows that the solution will diverge for arbitrary initial conditions. However, if the damping coefficient  $c$  is zero

$$a_{22} = a_{24} = a_{42} = a_{44} = 0$$

and the solution is stable for arbitrary displacements and zero velocities. Therefore, misalignment between the thrust vector and the vehicle center of mass at lift-off is not critical since the oscillations do not diverge. However, any disturbance in flight will result in divergent oscillations unless corrected by the pilot. An example is shown in figures 6 and 7.

The form of the response is determined by the denominator of equation (12), which is the characteristic equation. Therefore, the natural frequency of oscillation is dependent on the location of the roots of the quadratic factor. Since the location of the gimbal point will effect the value of  $(-a_{21} - a_{43})$ , as shown in equations (9) and (10), it will effect the location of the roots. Obviously, the magnitude and frequency of the response is a function of gimbal point location.

#### Pilot Describing Function

The control system can be characterized as a manual process for minimizing visually perceived errors by exercising continuous control so as to match visually presented input and output signals. The pilot closes the control loop by determining the error between a reference attitude and the present attitude; this process is called compensatory tracking.

The output of a nonlinear element (e.g., a human pilot) always contains higher harmonics, the effect of which depends on the characteristics of the other components. A quasi-linear equivalent to the nonlinear element, for a specific input, can be represented by a describing function (the equivalent linear element) and a remnant (the difference between the output of the nonlinear system and the equivalent linear element). Since components with relatively large inertia and long time constants filter higher harmonics considerably, a reasonable description for the pilot response characteristics is a linearized model of an equivalent transfer function which is linearly correlated with the system input.

To develop some criteria for stability of the controlled vehicle, it is necessary to have an expression for the control system. The pilot describing function has been used to express the functional relationship between the input and output of a single-axis manual control system (McRuer, et al.).

For this analysis, the input is the pitch attitude error

$$e = \theta_i - \theta \quad (13)$$

and the output is the pilot limb position (i.e., the angle between body 1 and body 2; the gimbal angle)

$$\delta = \theta - \beta \quad (14)$$

Thus, angular control of the vehicle in the pitch plane is represented by

$$Y_P = \frac{\delta}{e} = \frac{K_p (T_L s + 1) e^{-\tau s}}{(T_I s + 1) (T_{N1} s + 1) \left[ \left( \frac{s}{\omega_N} \right)^2 + \frac{2\zeta_N s}{\omega_N} + 1 \right]} \quad (15)$$

Since the parameters of the describing function are adjustable by the pilot, the function is only valid in the frequency domain and only exists under essentially stationary conditions.

Although the describing function represents a functional relationship, there appears to be some structural correlation. For example, the time delay  $e^{-\tau s}$  is due to excitation of the retina, nerve conduction, and data processing in the central nervous system. The neuromuscular system characteristic

$$(T_{N1} s + 1) \left[ \left( \frac{s}{\omega_N} \right)^2 + \frac{2\zeta_N s}{\omega_N} + 1 \right]$$

is compatible with physiological data for the muscle spindle and muscle tension, and the behavioral characteristics of the muscle/manipulator combination. The manipulator or control stick dynamics are not explicitly present in the term since virtually all experiments have been conducted with manipulators of negligible inertia, and the information which exists is insufficient to account for the effect analytically. However, the manipulator dynamics can be represented in terms of the gimbal angle and control force, using equations (6) and (14), as

$$\ddot{\delta} + (-a_{22} - a_{44})\dot{\delta} + (-a_{21} - a_{43})\delta = (a_{25} - a_{45})F_c$$

and in Laplace transform notation as

$$\frac{\delta}{F_c} = \frac{(a_{25} - a_{45})}{s^2 + (-a_{22} - a_{44})s + (-a_{21} - a_{43})} \quad (16)$$

The form of the manipulator dynamics is the same as the second-order component of the neuromuscular system. Since existing data support a first-order approximation to the neuromuscular system, the system is usually represented by  $T_N s + 1$  where

$$T_N \approx T_{N_1} + \frac{2\zeta_N}{\omega_N}$$

The factor

$$\frac{T_L s + 1}{T_I s + 1}$$

in a compensation or equalizing characteristic which, together with the gain  $K$ , is adjustable by the pilot to obtain system stability.



To simplify the subsequent analysis, the time delay in equation (15) can be replaced by a Páde approximation

$$e^{-\tau s} \approx \left( \frac{\frac{-\tau}{2} s + 1}{\frac{\tau}{2} s + 1} \right)$$

The pilot describing function then becomes

$$Y_p = \frac{\delta}{e} \approx \frac{K_p (T_L s + 1) \left( -\frac{\tau}{2} s + 1 \right)}{(T_I s + 1) (T_N s + 1) \left( \frac{\tau}{2} s + 1 \right)} \quad (17)$$

#### Attitude Control System Stability

From equations (6) and (14), the linear equation for vehicle pitch attitude  $\theta$  can be written in terms of the gimbal angle  $\delta$  as

$$\ddot{\theta} = a_{21} \delta + a_{22} \dot{\delta} \quad (18)$$

Taking the Laplace transform and rearranging, the transfer function for the vehicle dynamics is

$$Y_c = \frac{\theta}{\delta} = \frac{a_{22}s + a_{21}}{s^2} \quad (19)$$

The product of the transfer function for the control system (eq. (17)) and the transfer function for the vehicle dynamics yields the system open-loop transfer function

$$G(s) = Y_p Y_c = \frac{\theta}{e} = \frac{K_p (T_L s + 1) \left( -\frac{\tau}{2} s + 1 \right) (a_{22}s + a_{21})}{s^2 (T_I s + 1) (T_N s + 1) \left( \frac{\tau}{2} s + 1 \right)} \quad (20)$$

and the closed-loop transfer function is

$$\frac{G(s)}{1 + G(s)} = \frac{\theta}{\theta_i} = \frac{K_p(T_L s + 1)\left(-\frac{\tau}{2}s + 1\right)(a_{22}s + a_{21})}{s^2(T_I s + 1)(T_N s + 1) + K_p(T_L s + 1)\left(-\frac{\tau}{2}s + 1\right)(a_{22}s + a_{21})} \quad (21)$$

The system characteristic equation is

$$\begin{aligned} \frac{\tau}{2} T_I T_N s^5 + \left[ \frac{\tau}{2} (T_I + T_N) + T_I T_N \right] s^4 + \left[ \frac{\tau}{2} (1 - K_p a_{22} T_L) + T_I + T_N \right] s^3 + \left\{ 1 + K_p \left[ a_{22} \left( T_L - \frac{\tau}{2} \right) - \frac{\tau}{2} a_{21} T_L \right] \right\} s^2 \\ + K_p \left[ a_{22} + a_{21} \left( T_L - \frac{\tau}{2} \right) \right] s + K_p a_{21} \end{aligned} \quad (22)$$

The Hurwitz algorithm determines the criteria which the system characteristic equation must satisfy for stability. The Hurwitz matrix for a fifth-order system is

$$H = \begin{bmatrix} b_1 & b_3 & b_5 & 0 & 0 \\ 1 & b_2 & b_4 & 0 & 0 \\ 0 & b_1 & b_3 & b_5 & 0 \\ 0 & 1 & b_2 & b_4 & 0 \\ 0 & 0 & b_1 & b_3 & b_5 \end{bmatrix}$$

where

$$\begin{aligned}
 b_0 &= \frac{\tau}{2} T_I T_N \\
 b_1 &= \frac{\left[ \frac{\tau}{2} (T_I + T_N) + T_I T_N \right]}{b_0} \\
 b_2 &= \frac{\left[ \frac{\tau}{2} (1 - K_p a_{22} T_L) + T_I + T_N \right]}{b_0} \\
 b_3 &= \frac{\left\{ 1 + K_p \left[ a_{22} \left( T_L - \frac{\tau}{2} \right) - \frac{\tau}{2} a_{21} T_L \right] \right\}}{b_0} \\
 b_4 &= \frac{K_p \left[ a_{22} + a_{21} \left( T_L - \frac{\tau}{2} \right) \right]}{b_0} \\
 b_5 &= \frac{K_p a_{21}}{b_0}
 \end{aligned} \tag{23}$$

A polynomial whose zeros all have negative real parts is called a Hurwitz polynomial. The necessary and sufficient condition for equation (22) to be a Hurwitz polynomial is that the principal minors of  $H$  all be positive.

The principal minors of  $H$  are

$$H_1 = b_1$$

$$H_2 = b_1 b_2 - b_3$$

$$H_3 = (b_1 b_2 - b_3) b_3 - (b_1 b_4 - b_5) b_1$$

$$H_4 = \left[ (b_1 b_2 - b_3) b_3 - (b_1 b_4 - b_5) b_1 \right] b_4 - \left[ (b_1 b_2 - b_3) b_2 - (b_1 b_4 - b_5) \right] b_5$$

$$H_5 = \left\{ \left[ (b_1 b_2 - b_3) b_3 - (b_1 b_4 - b_5) b_1 \right] b_4 - \left[ (b_1 b_2 - b_3) b_2 - (b_1 b_4 - b_5) \right] b_5 \right\} b_5$$

From equation (23) it can be seen that

$$b_0 > 0$$

$$b_1 > 0$$

since all the parameters are positive.

Therefore

$$H_1 = b_1 > 0$$

The remaining principal minors, which must be positive, are

$$H_2 = \frac{\frac{\tau^2}{4} (T_I + T_N) + \frac{\tau}{2} (T_I^2 + T_N^2) + T_{IN}^T (T_I + T_N + \tau) + K_p \left\{ a_{22} \left[ \frac{T_{IN}^T}{4} (\frac{\tau^2}{4} - \tau T_L) - \frac{\tau^2}{4} (T_I + T_N) T_L \right] + a_{21} \frac{\tau^2}{4} T_{IN}^T T_L \right\}}{\left( \frac{\tau}{2} T_{IN}^T \right)^2} \quad (24)$$

$$H_3 = \frac{H_2 \left\{ 1 + K_p \left[ a_{22} \left( T_L - \frac{\tau}{2} \right) - a_{21} \frac{\tau}{2} T_L \right] \right\}}{\frac{\tau}{2} T_{IN}^T} + \frac{\left[ \frac{\tau}{2} (T_I + T_N) + T_{IN}^T \right] K_p}{\left( \frac{\tau}{2} T_{IN}^T \right)^3} \left\{ a_{21} \left[ T_{IN}^T (\tau - T_L) + \frac{\tau}{2} (T_I + T_N) \left( \frac{\tau}{2} - T_L \right) \right] \right\} - a_{22} \left[ \frac{\tau}{2} (T_I + T_N) + T_{IN}^T \right] \quad (25)$$

$$H_4 = \frac{K_p}{\left( \frac{\tau}{2} T_{IN}^T \right)^2} \left\{ H_3 \frac{\tau}{2} T_{IN}^T \left[ a_{22} + a_{21} \left( T_L - \frac{\tau}{2} \right) \right] - H_2 a_{21} \left[ \frac{\tau}{2} \left( 1 - K_p a_{22} T_L \right) + T_I + T_N \right] \right\} + \frac{K_p^2 a_{21}}{\left( \frac{\tau}{2} T_{IN}^T \right)^3} \left\{ a_{22} \left[ \frac{\tau}{2} (T_I + T_N) + T_{IN}^T \right] \right. \\ \left. + a_{21} \left[ (T_I + T_N) \left( T_L - \frac{\tau}{2} \right) \frac{\tau}{2} + T_{IN}^T (T_L - \tau) \right] \right\} \quad (26)$$

$$H_5 = \frac{H_4 K_p a_{21}}{\frac{\tau}{2} T_{IN}^T} \quad (27)$$

The equations for the principal minors indicate that  $H_2$ ,  $H_3$ ,  $H_4$ , and  $H_5$  could be positive or negative depending on the sign and magnitude of  $a_{21}$  and  $a_{22}$ . The parameters  $T_L$ ,  $T_I$ , and  $K_p$  must be adjusted by the pilot to insure that the minors are positive, thus rendering the system stable. For a particular set of vehicle parameters and values of  $T_N$  and  $\tau$ , the inequalities for  $H_n$  may be solved to obtain the permissible range of values of  $T_L$ ,  $T_I$ , and  $K_p$ .

The impulse response of the attitude control system closed-loop transfer function is shown in figure 8 for a set of parameters which satisfy Hurwitz criteria and in figure 9 for a set which do not.

## CONCLUSIONS

The autonomous vehicle is unstable except for the special case of zero damping and initial conditions on displacement only. The magnitude and frequency of the response differ for the configurations with the gimbal point above or below the center of mass since the vehicle parameters which define the configuration determine the location of the roots of the characteristic equation.

Kinesthetic control allows less freedom in design because the physical parameters of the pilot (i.e., mass, inertia, length) cannot be adjusted to meet the stability criteria.

The stability criteria for the vehicle and for the closed-loop system should be useful in defining the permissible range of vehicle parameters for stable operation. Extension of this procedure to multiaxis control will be complicated by vehicle axis coupling through products of inertia and by the requirement to generate different equilibrations in different axes simultaneously.

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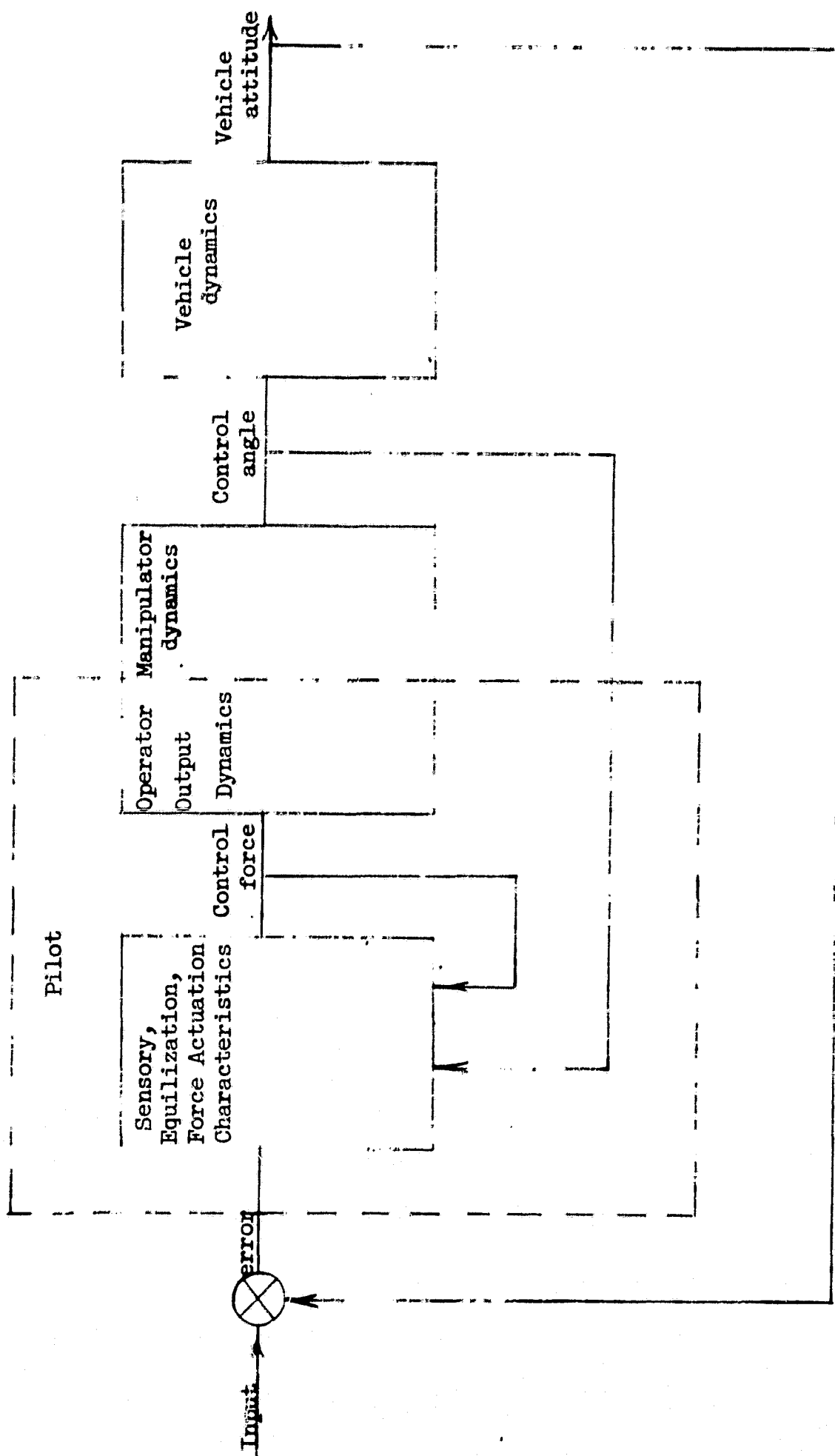


Figure 1.- Single-axis attitude control system block diagram.

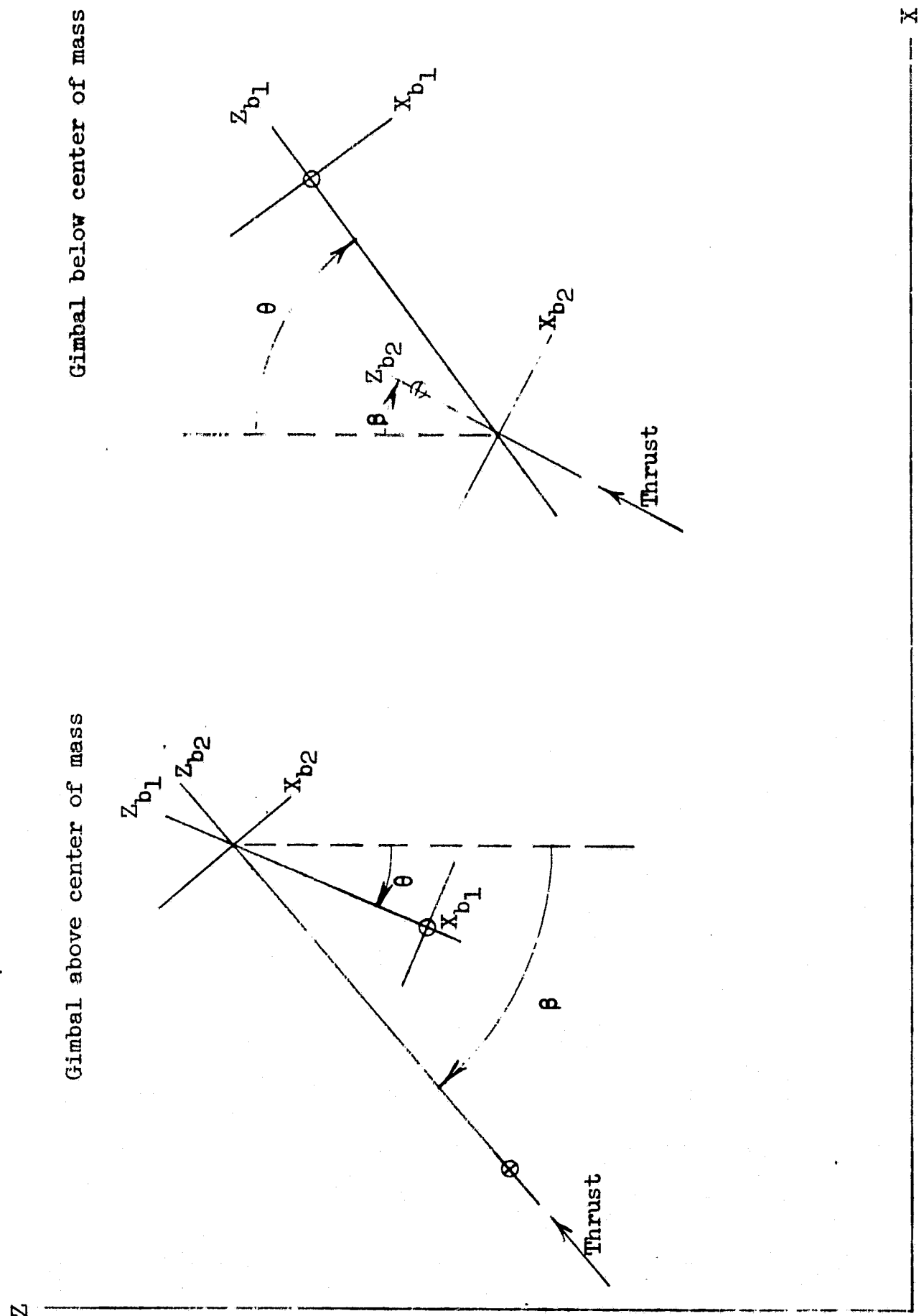


Figure 2.- Coordinate system.

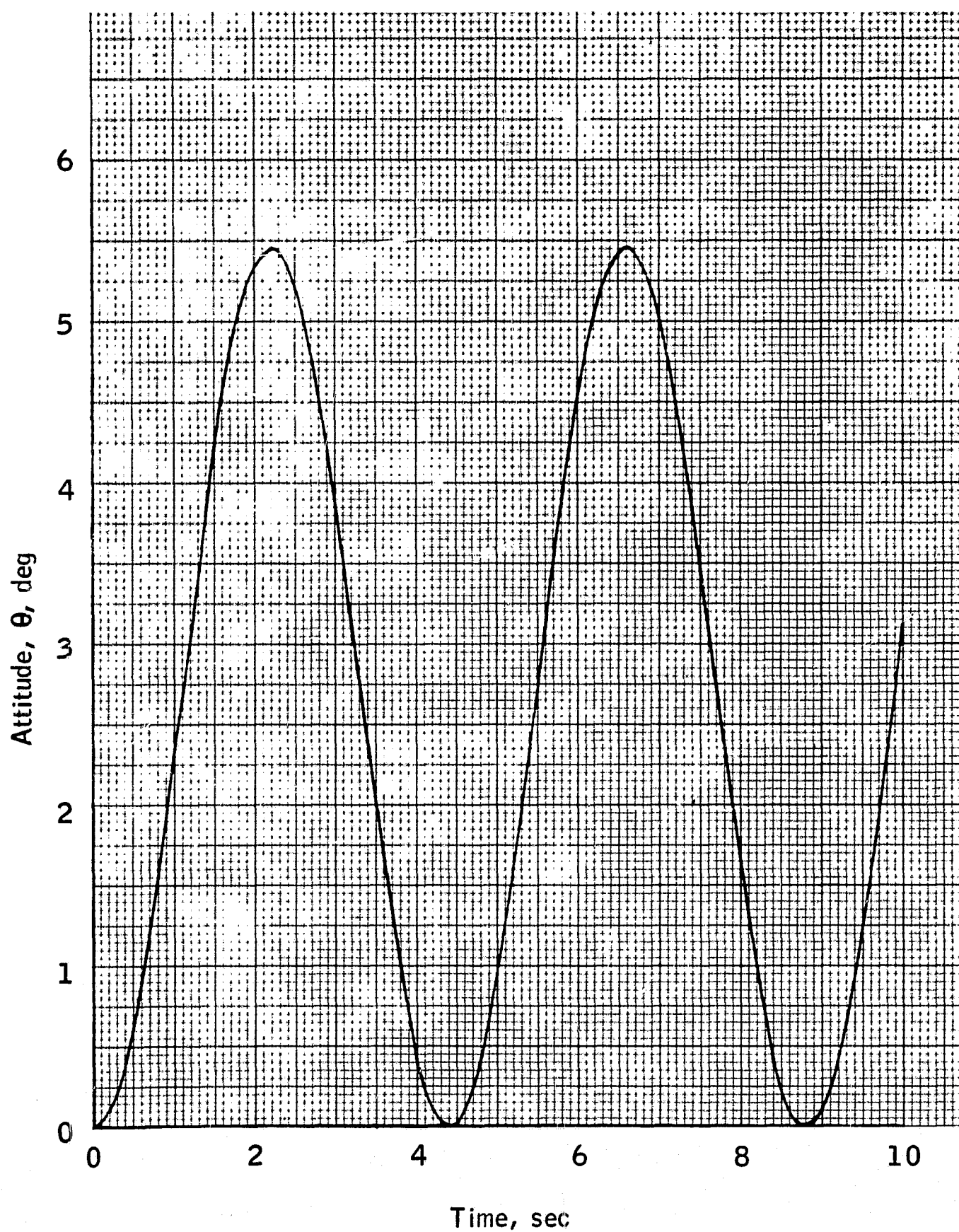


Figure 3.- Numerical solution of nonlinear equations of motion.

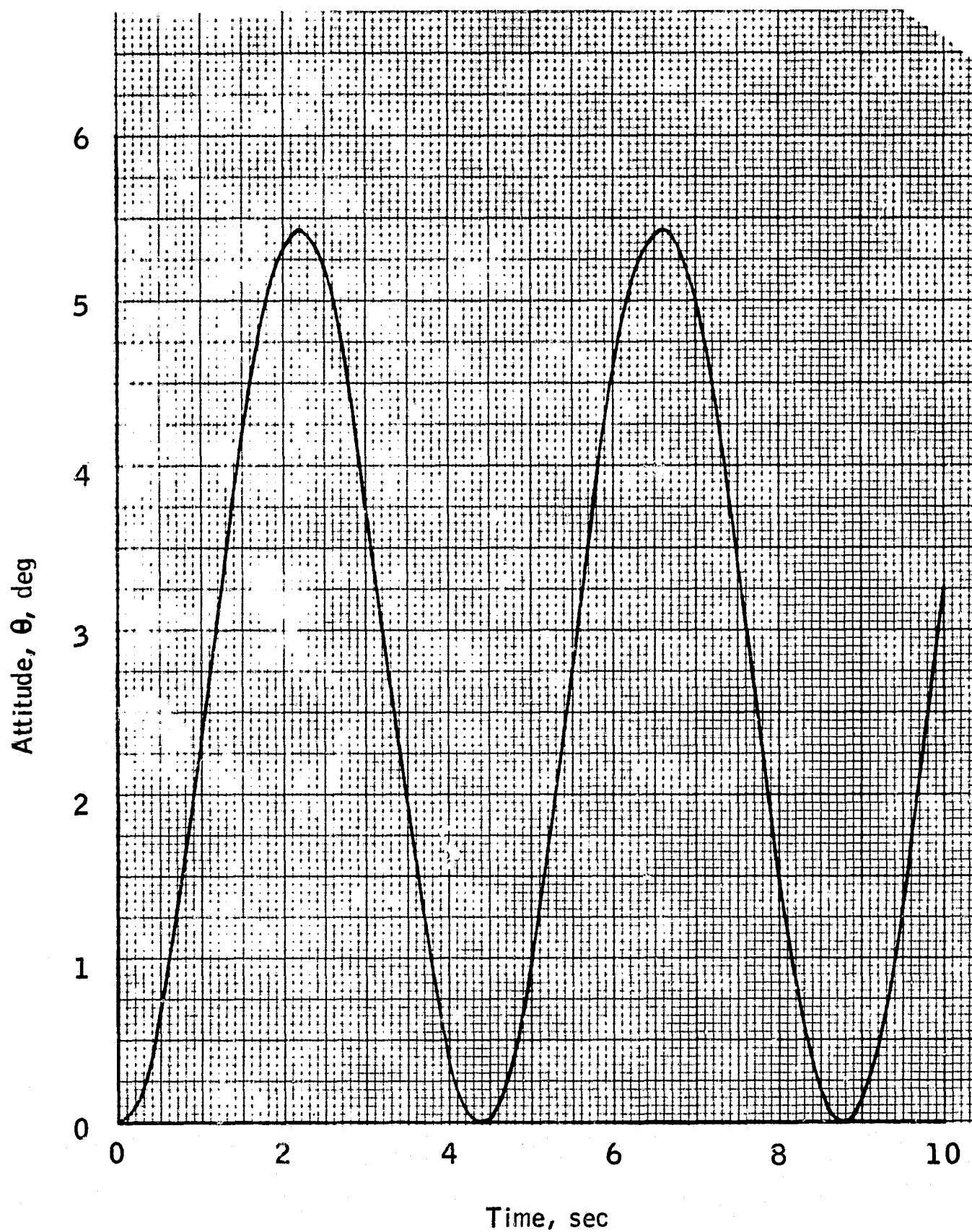


Figure 4.- Numerical solution of linear equations of motion.

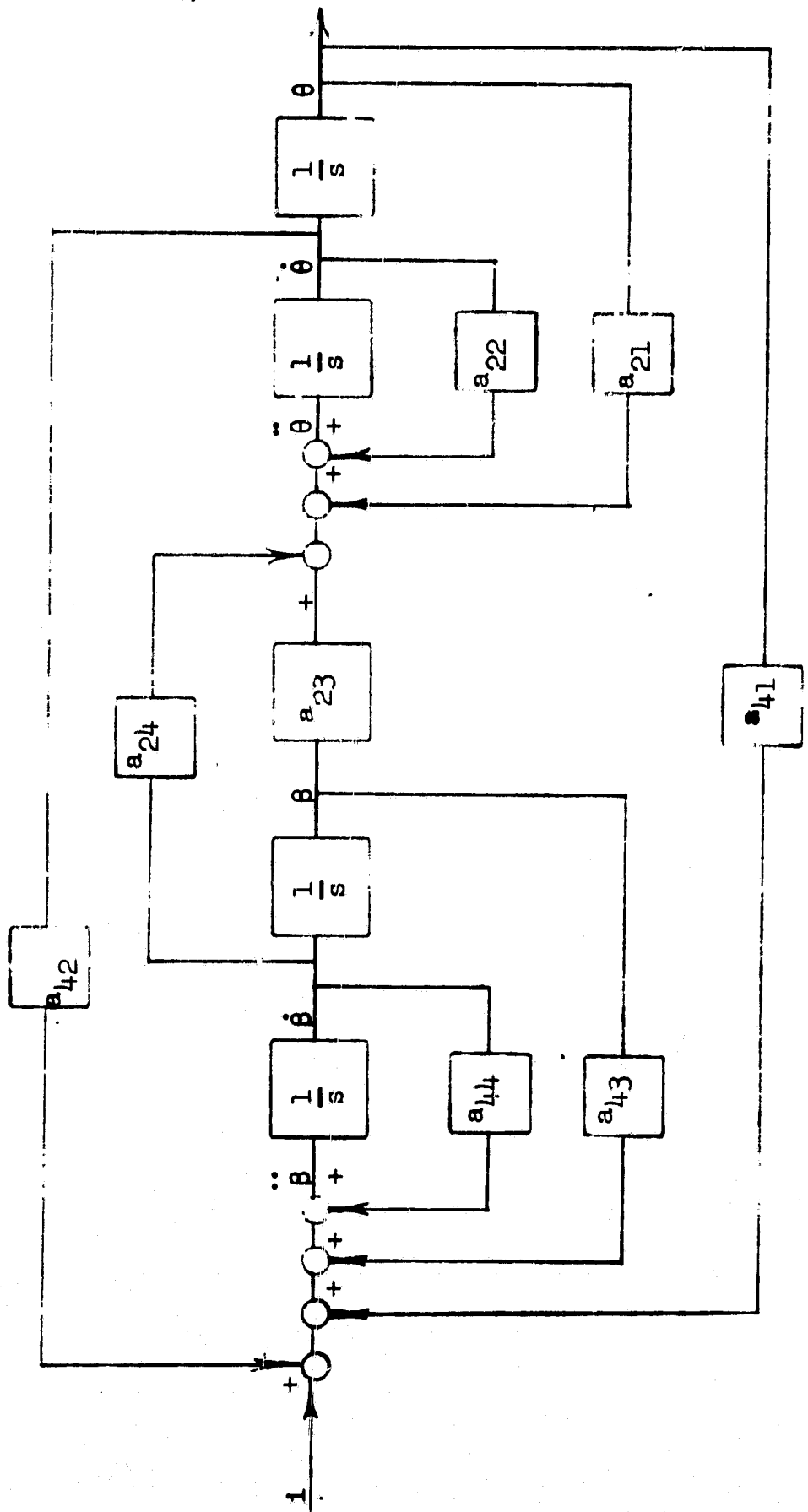


Figure 5.- Block diagram of vehicle dynamics.



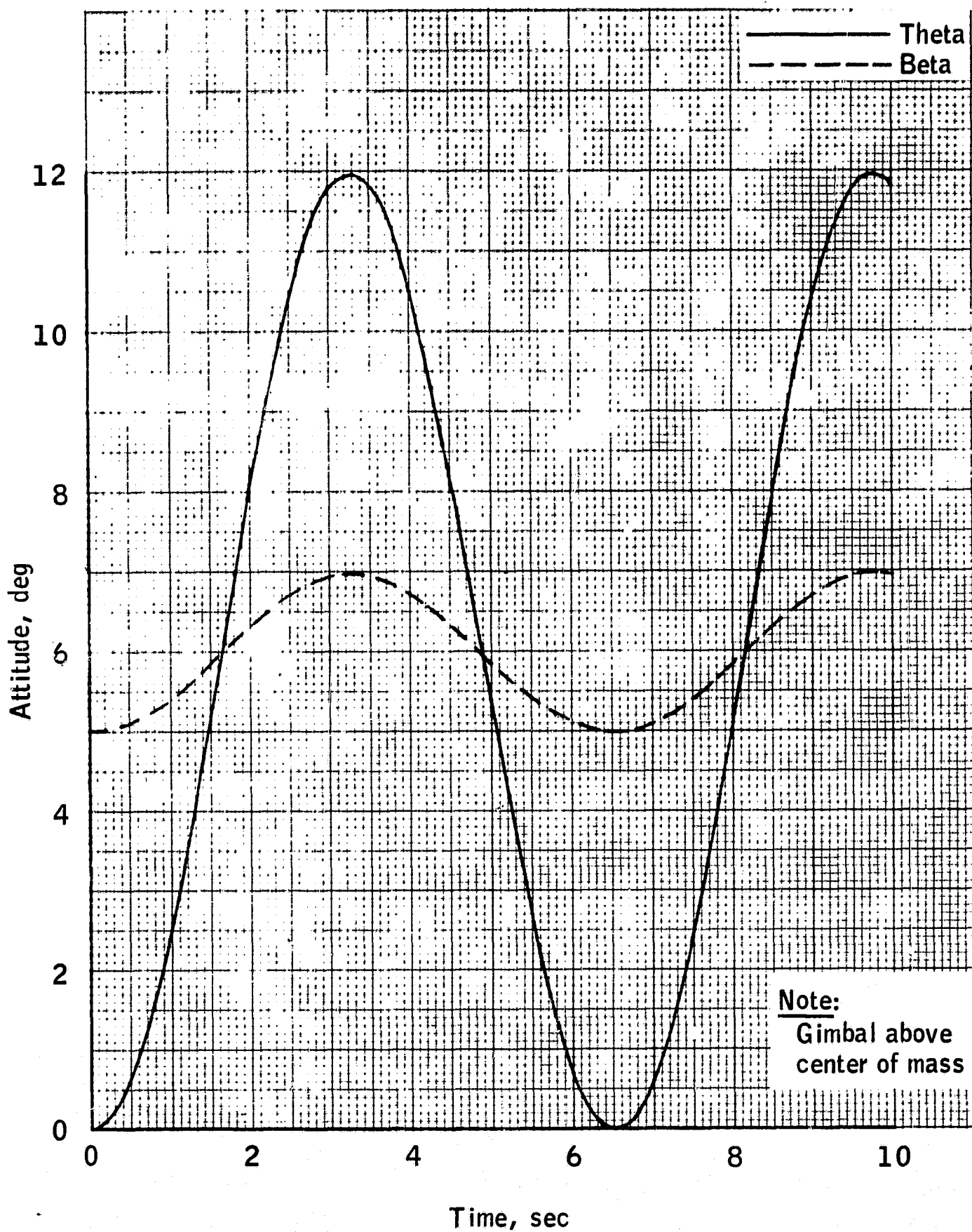


Figure 6.- Response of thrust vector controlled vehicle (autonomous, gimbal above center of mass).

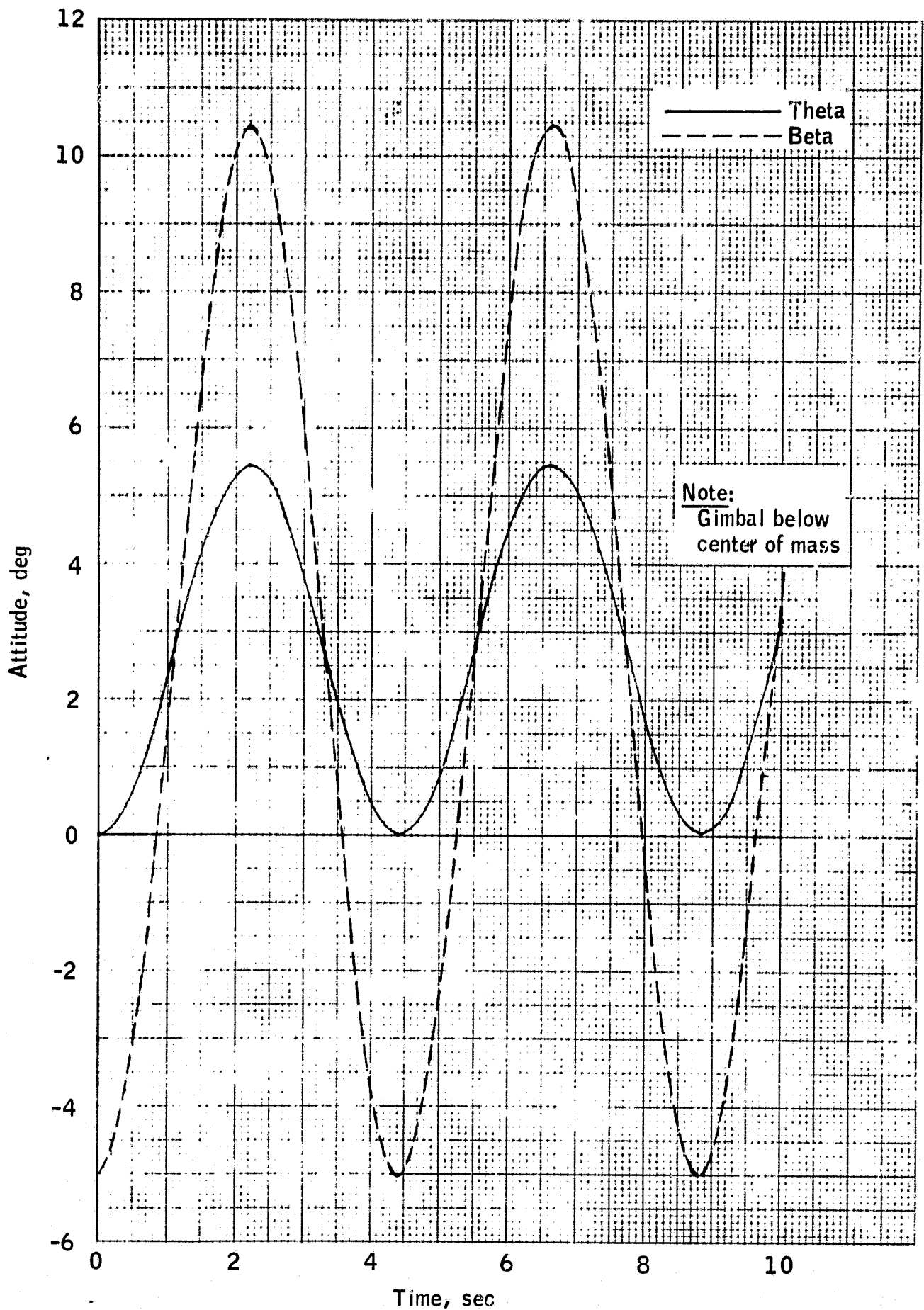


Figure 7.- Response of thrust vector controlled vehicle (autonomous, gimbal below center of mass).

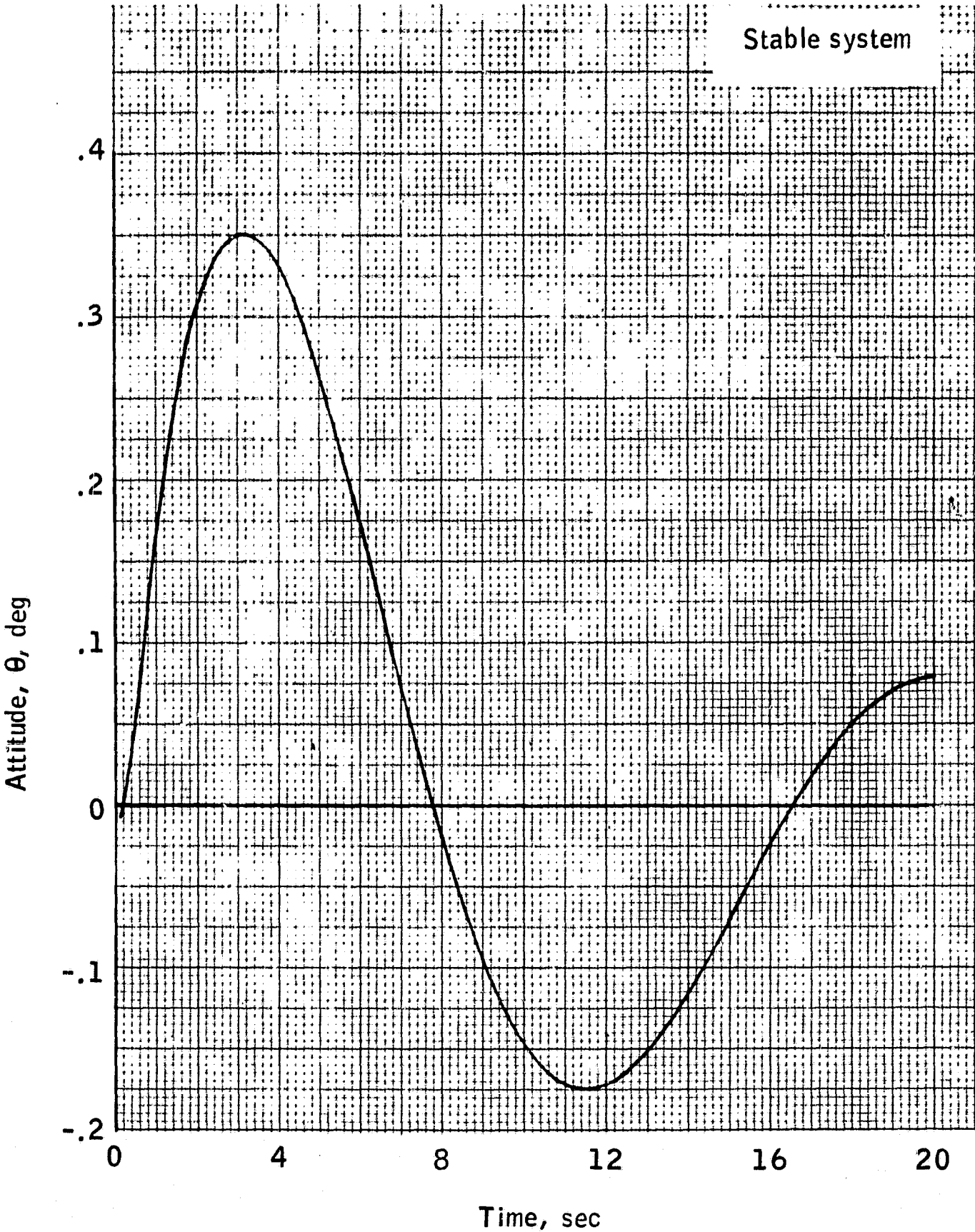


Figure 8.- Impulse response for attitude control system (stable).

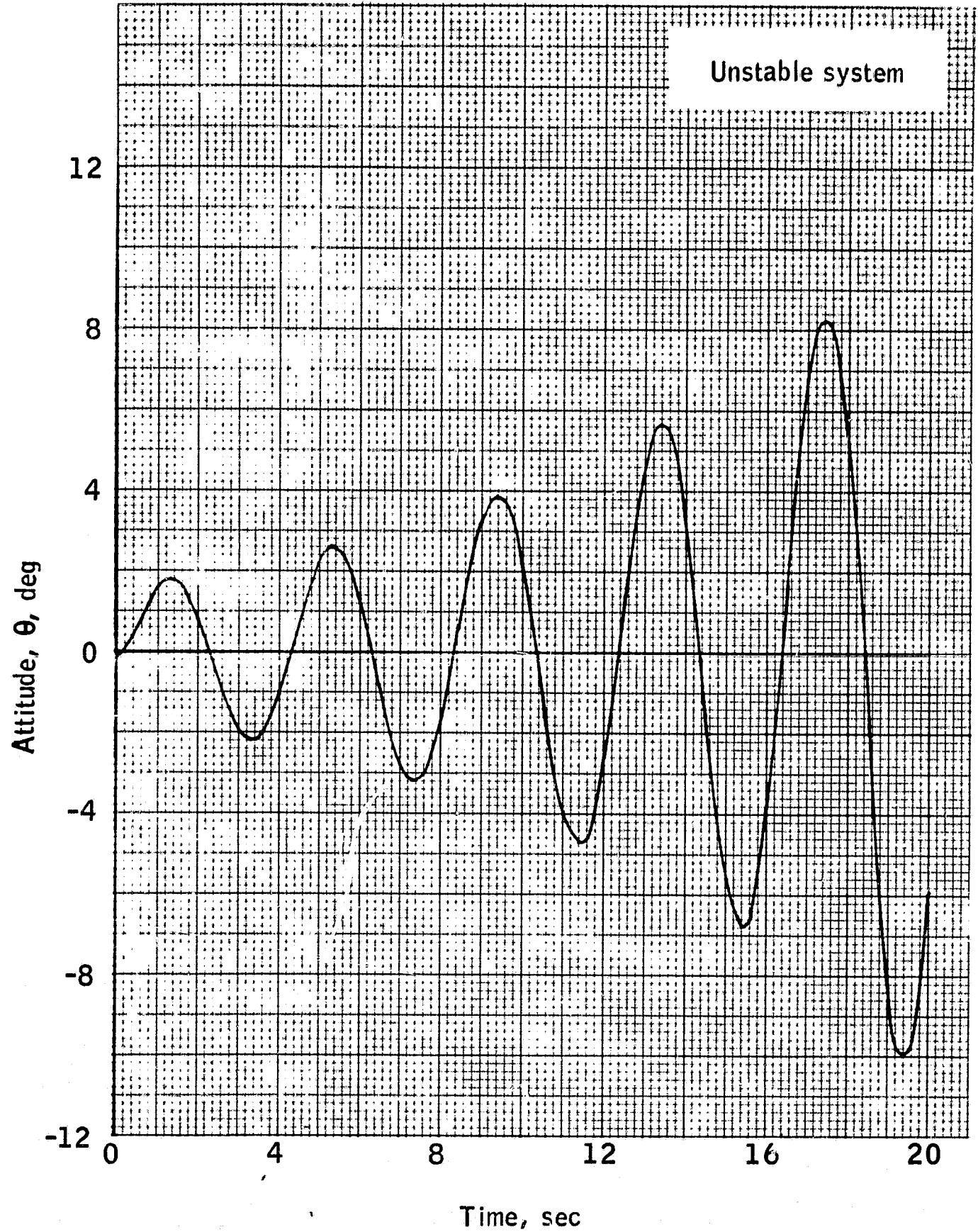


Figure 9.- Impulse response for attitude control system (unstable).